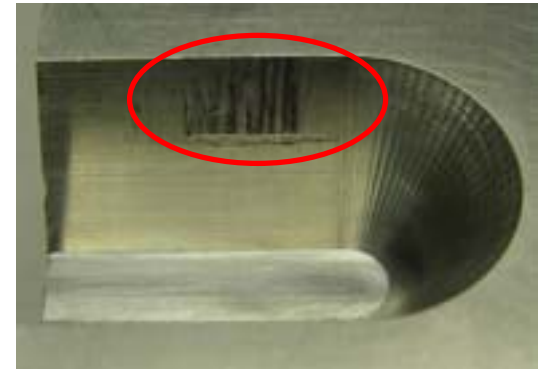


# **USING CONSTRAINTS IN MECHANISM DESIGN**

# Alternatives to motion with physical contact

## Problems you can not avoid with contact:

- ⊙ Surface topology (finish)
- ⊙ Wear and Fretting
- ⊙ Friction
- ⊙ Limited resolution, at best on order of microns....



Wear on Groove

## Next generation applications require nanometer level fixtures, i.e.:

- ⊙ Fiber optics
- ⊙ Photolithography

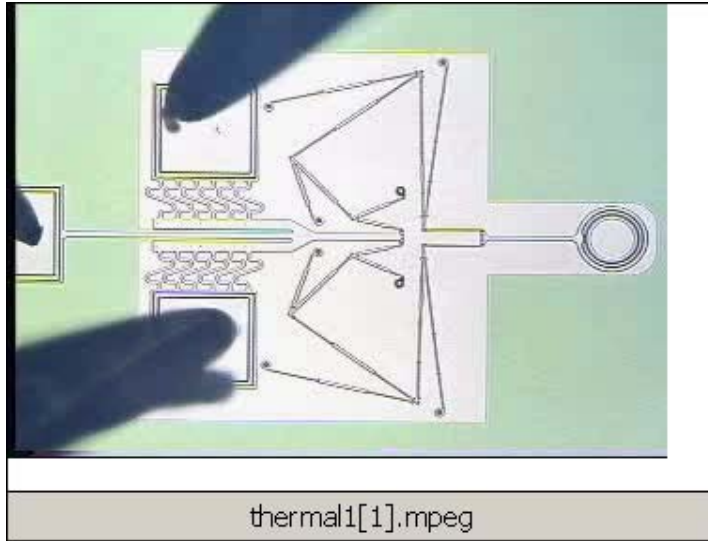
## Compliant mechanisms:

- ⊙ Mechanical reduction to interface with larger scale actuators
- ⊙ Motion through strain
- ⊙ Small and moderately sized motions in comparison to mechanism size
- ⊙ Can be made to emulate machines

# Compliant mechanism examples

University of Michigan: Prof. Sridhar Kota

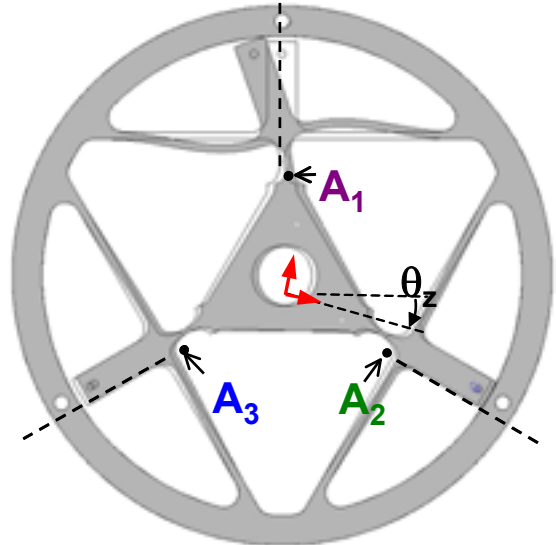
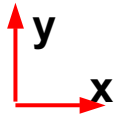
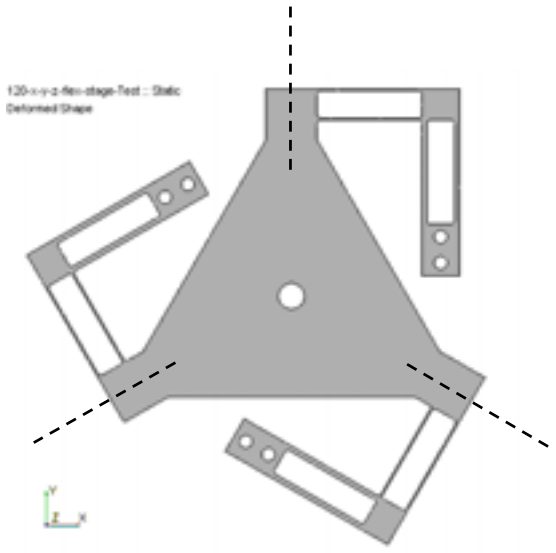
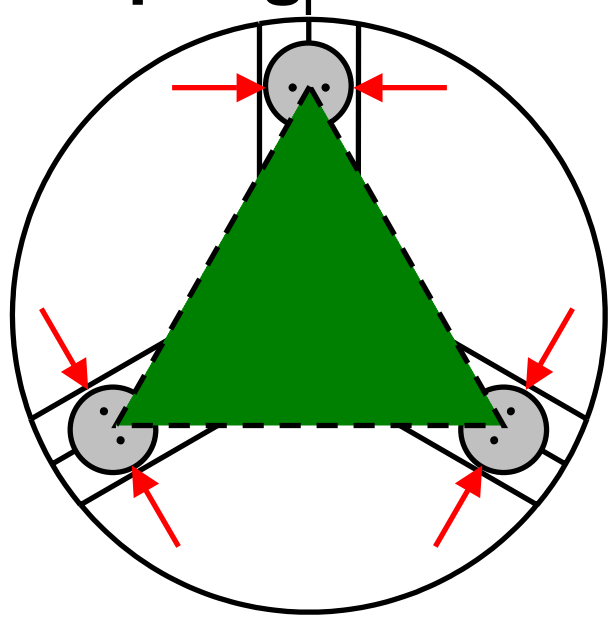
- ⦿ <http://www.engin.umich.edu/labs/csdl/index.htm>



## Why compliant mechanisms in precision fixtures

- ⦿ Repeatable/low hysteresis
- ⦿ No assembly
- ⦿ No contact

# From kinematic couplings to compliant stages

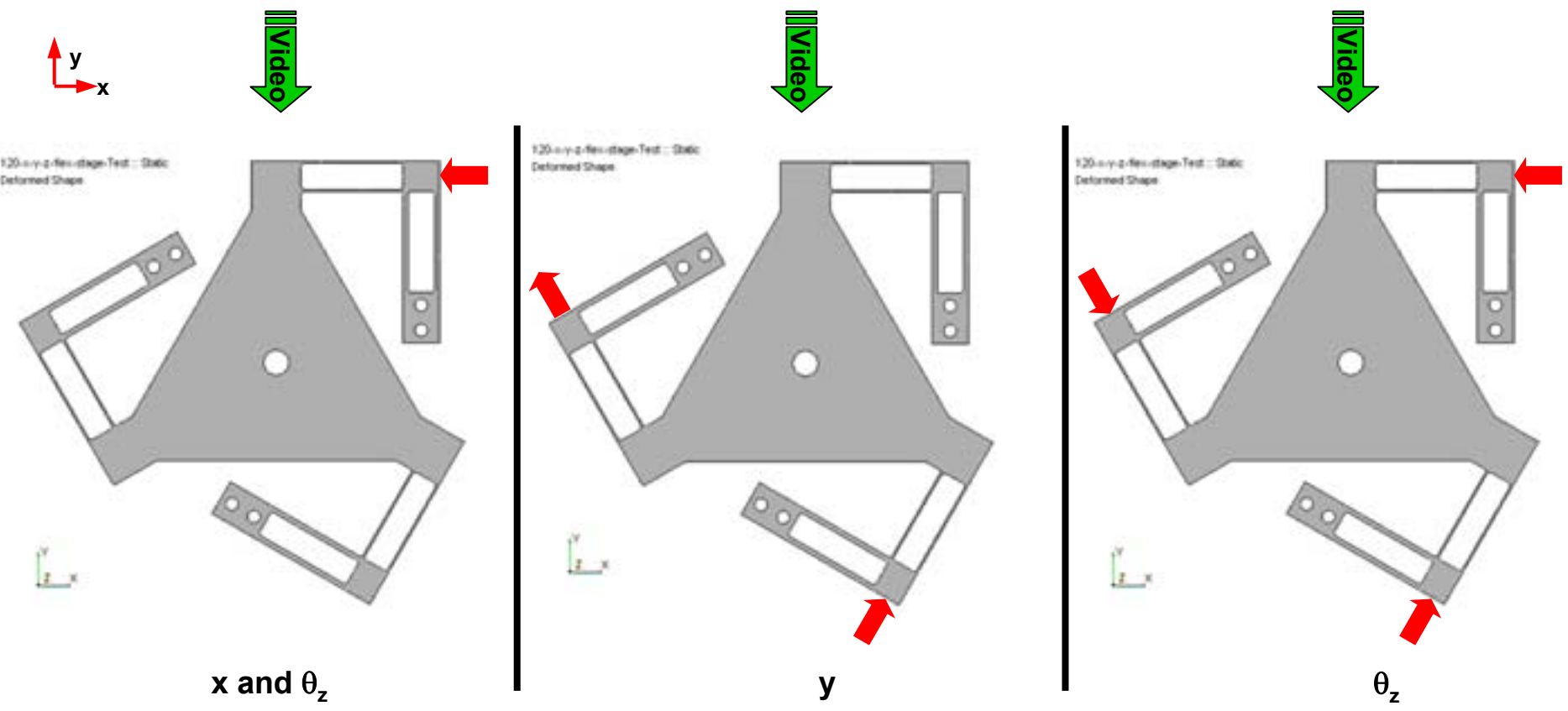


# Constraint based compliant mechanisms

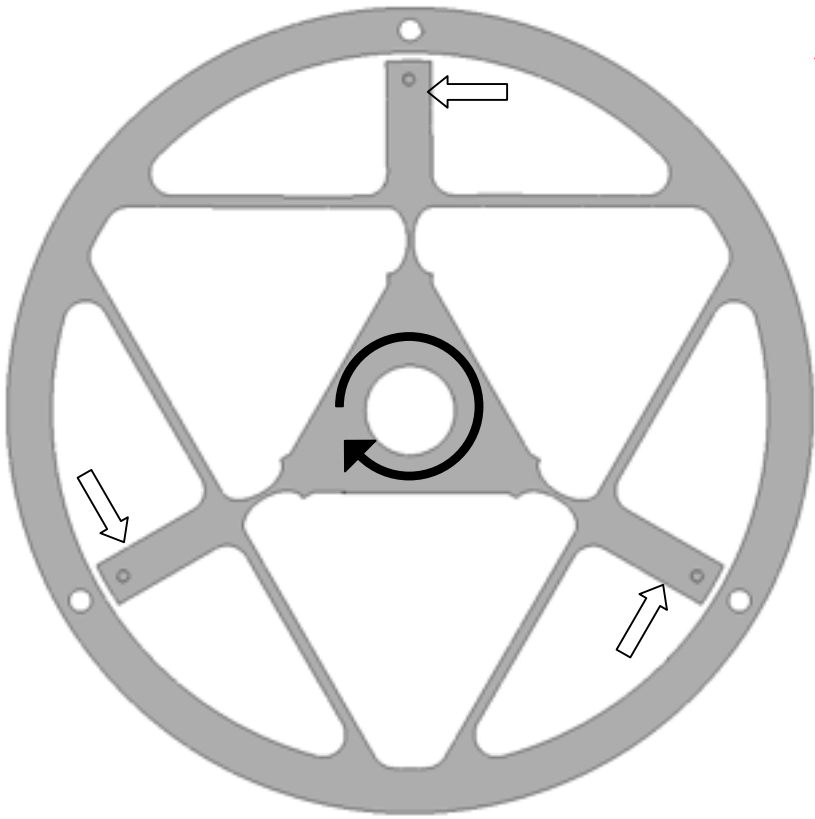
High volume, low cost, multi-degree of freedom alignment

Example 3 DOF flexure system:

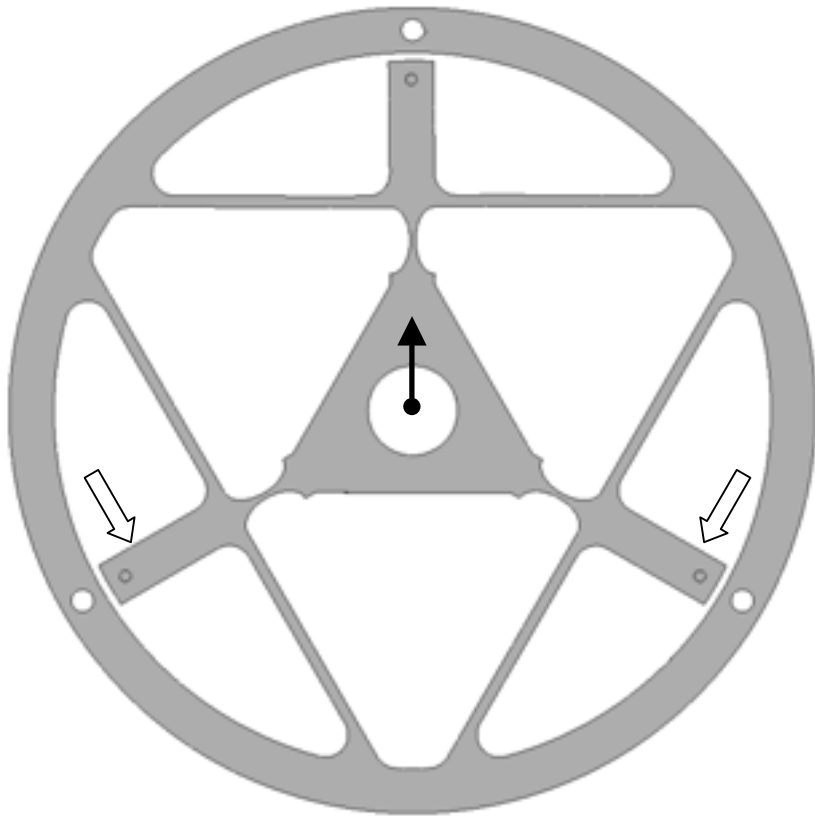
Target applications: Opto-electronic packaging/alignment



# Constraint based compliant mechanisms cont.



$\theta_z$

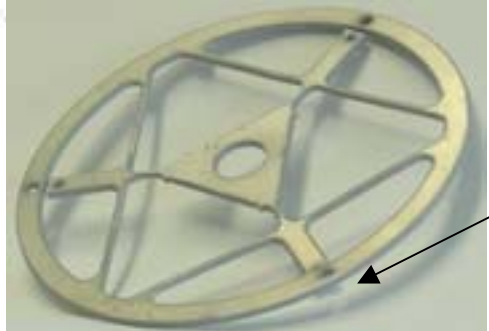
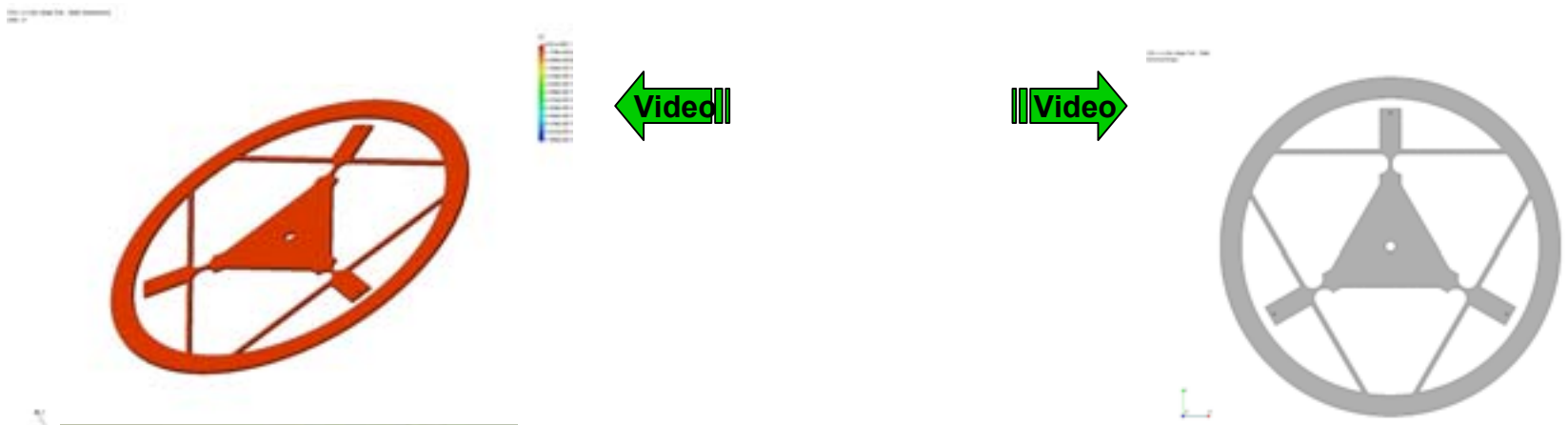


$y$

# Constraint based compliant mechanisms cont.

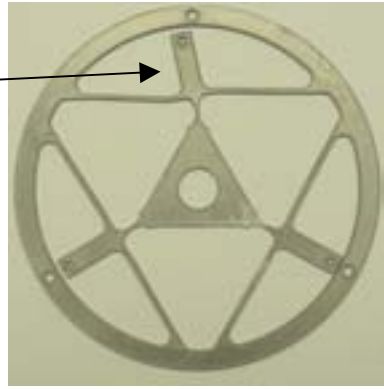
## Example 6 DOF alignment capability

Target app.: Micro and meso scale positioning (i.e. opto-electronics)



Permanent set due to plastic deformation

Have option to rely on elastic deformation



$x$  and  $\theta_z$

$z, \theta_x, \theta_y$

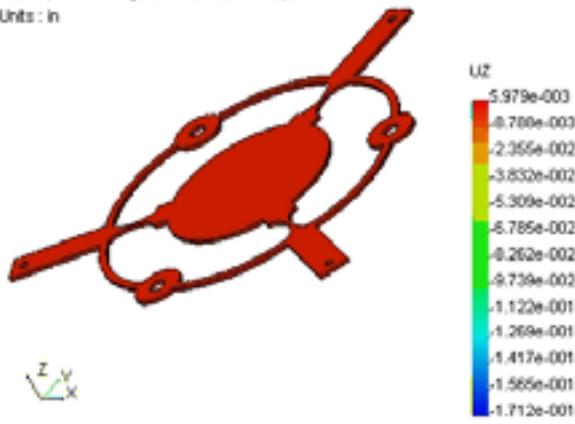
# Constraint based compliant mechanisms cont.

## Example 6 DOF alignment capability

Target app.: Micro/meso scale positioning (I.e. opto-electronics)



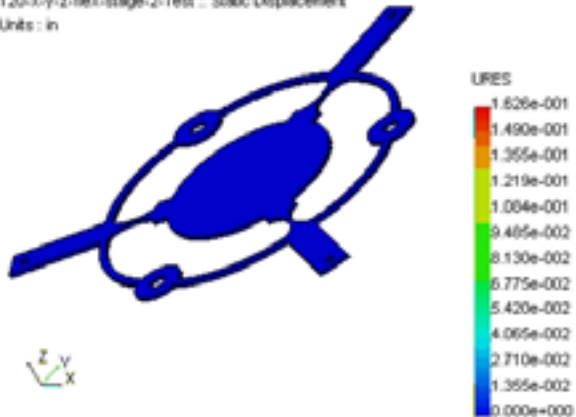
120-x-y-z-flex-stage-2-Test :: Static Displacement  
Units: in



$z, \theta_x, \theta_y$



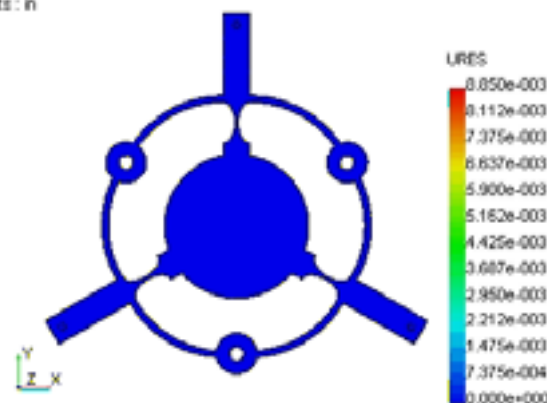
120-x-y-z-flex-stage-2-Test :: Static Displacement  
Units: in



$z$



120-x-y-z-flex-stage-2-Test :: Static Displacement  
Units: in



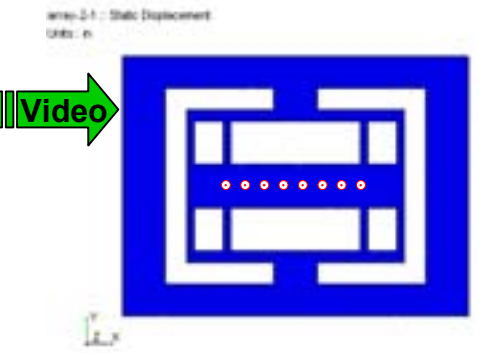
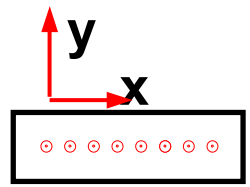
$\theta_z$



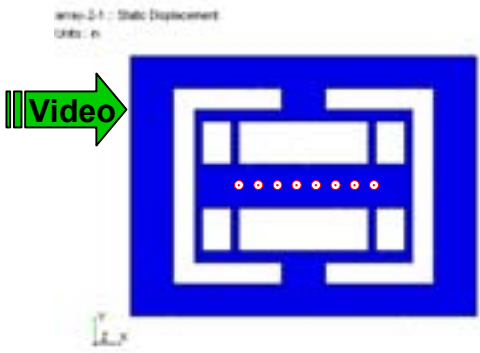
# Constraint based compliant mechanisms cont.

3 DOF active alignment [ x, y, z ] & 2 DOF passive alignment [ z,  $\theta_y$  ]

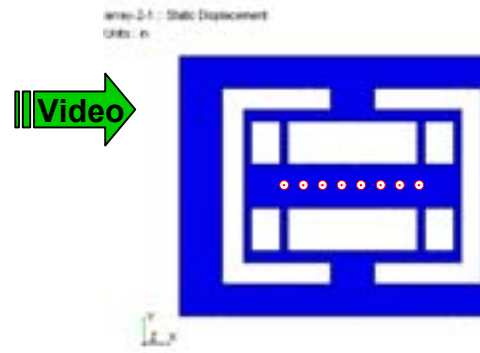
Good fit for wire-EDM (stacked sheets) ~ order of \$ 1 - 10



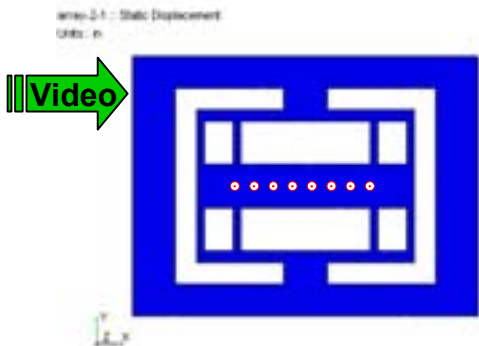
$\Delta x$



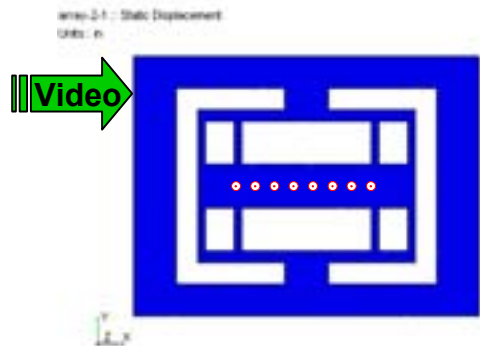
$\Delta y$



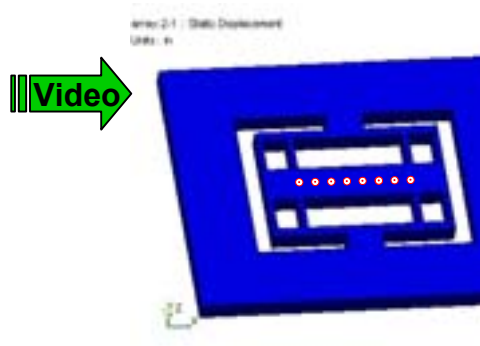
$\Delta \theta_z$



$\Delta x \ \& \ \Delta \theta_z$



$\Delta x \ \& \ \Delta y$

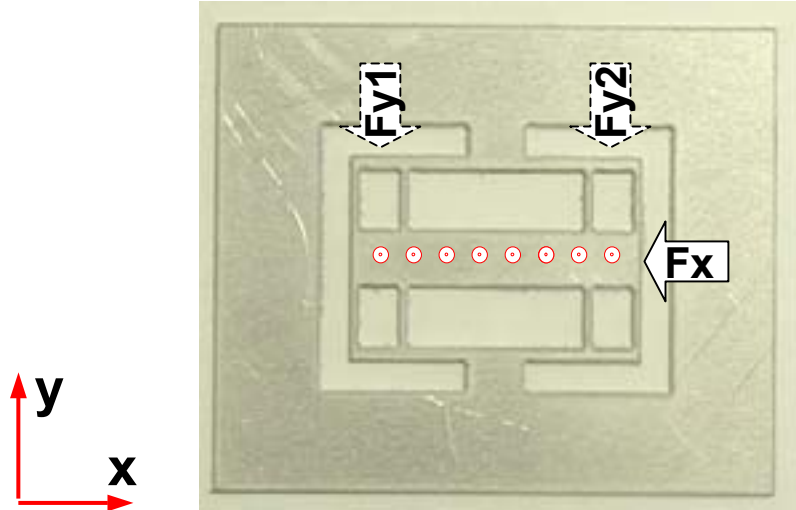


$\Delta \theta_y$

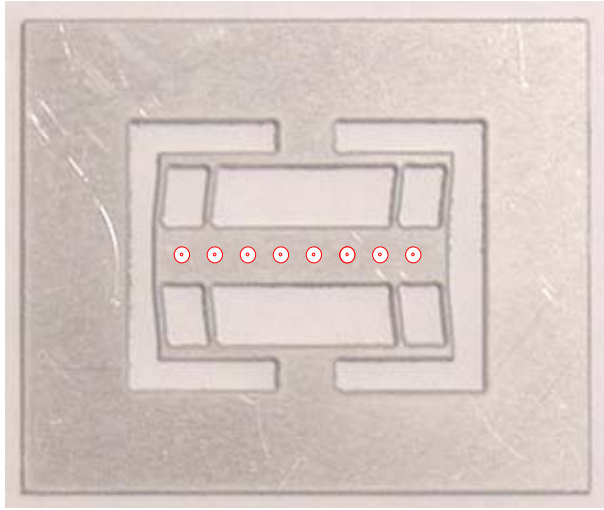
# Constraint based compliant mechanisms cont.

Plastic deformation can be utilized for position keeping

Device should be potted in place to avoid stress relief



Initial position



Plastically flexed

# Constraint based compliant mechanisms cont.

Static or flexible kinematic coupling

Components biased toward each other

Flexure takes up bias, provides mating force in z direction

